Mechanical behaviour analysis of FGM plates on elastic foundation using a new exponential-trigonometric HSDT.

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Abstract. In this research, a new two-dimensional (2D) and quasi three-dimensional (quasi-3D) higher order shear deformation theory is devised to address the bending problem of functionally graded plates resting on an elastic foundation. The displacement field of the suggested theories takes into account a parabolic transverse shear deformation shape function and satisfies shear stress free boundary conditions on the plate surfaces. It is expressed as a combination of trigonometric and exponential shear shape functions. The Pasternak mathematical model is considered for the elastic foundation. The material properties vary constantly across the FG plate thickness using different distributions as power-law, exponential and Mori–Tanaka model. By using the virtual works principle and Navier's technique, the governing equations of FG plates exposed to sinusoidal and evenly distributed loads are developed. The effects of material composition, geometrical parameters, stretching effect and foundation parameters on deflection, axial displacements and stresses are discussed in detail in this work. The obtained results are compared with those reported in earlier works to show the precision and simplicity of the current formulations. A very good agreement is found between the predicted results and the available solutions of other higher order theories. Future mechanical analyses of three-dimensionally FG plate structures can use the study's findings as benchmarks.

Keywords: Bending; Stress; Functionally graded plate; Shear deformation theory; Stretching effect, Winkler-Pasternak parameters.

1. Introduction

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Functionally graded materials (FGMs) are a brand-new class of inhomogeneous composite materials that are created by combining metals and ceramics. These materials have incredible effectiveness since the mechanical properties of FGMs vary constantly along the thickness direction (Barretta *et al*. 2016, Rachid *et al*. 2022). These two material characteristics vary continuously and smoothly in the desired directions. FGM can thus not only combine the best qualities of ceramic and metal but also reduce residual and thermal stresses, both of which are quite frequent in traditional multi-layered composite systems (Mengzhen *et al*. 2021).

Owing to this characteristic, FGM have been investigated in numerous engineering applications including the aerospace, automotive, aviation and defence sectors, as well as more recently the electronic and biological applications (Nguyen 2015, Abdelhak 2016, Aldousari 2017). Due to the widespread use of FGM in engineering domains recently, a number of theories have been established

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to analyze the bending, buckling, and dynamic behaviors of diverse FG structures (Hebbar *et al*. 2020, Arefi and Allam 2015, Abdulrazzaq *et al*. 2020, Zaoui *et al.* 2021, 2022a). In order to study the free vibration of FG rectangular plates resting on elastic foundations, Mantari *et al*. (2014) introduced a new HSDT. Xiang and Kang (2013) carried out the bending analysis of functionally graded plates using an nth-order shear deformation theory. Based on an improved third-order shear deformation theory, Le *et al*. (2020) developed finite element formulations for analysing the vibration characteristics of FGSW plates partially supported by Pasternak foundation. For studying the static response of functionally graded plates resting on a Winkler-Pasternak foundation under a transverse uniform load or a sinusoidal distributed force, Benyoucef *et al*. (2010) introduced a novel hyperbolic shear deformation theory of plates. Using fewer unknowns, Thai and Kim (2013) used quasi-3D sinusoidal shear deformation theory to investigate the bending behaviour of FGM plates. The impact of porosity was studied utilizing the four variable plate theory for P- FGM plates with sinusoidal shape function. Based on Reddy's third-order shear deformation theory, Hosseini-Hashemi *et al*. (2011) introduced a novel exact closed-form approach to solve free vibration analysis of FG rectangular thick plates. Carrera *et al*. (2011) investigated the effects of thickness stretching in functionally graded plates and shells. Neves *et al*. suggested using quasi-3D sinusoidal and hyperbolic shear deformation theory to investigate the static and free vibration responses of FG plates (2012a, 2012b). Thai *et al*. (2013) investigated the bending, buckling and free vibration of FG thick plates resting on elastic foundation using a simple refined higher-order shear deformation theory with two variables. Jha *et al*. (2013) examined the free vibration behaviour of FG elastic, rectangular and simply supported plates using higher order shear/shear-normal deformation theories. Ouinas and Achour (2013) used the finite element approach to examine the buckling behavior of a square plate made of composite laminated material with an elliptical notch. Mantari and Soares (2015) presented the static response of advanced composite plates and shells utilizing quasi-3D non-polynomial sinusoidal higher-order shear deformation theories. Zhang *et al*. (2014) established a 3D elasticity solution for the bending of thick FG plates utilizing a hybrid semi-analytical approach—the state-space based differential quadrature method. Al-Khateeb and Zenkour (2014) used a revised shear and normal deformations plate theory to examine the effects of temperature and moisture on the bending behaviour of FG plates resting on elastic foundations. Mantari and Soares (2013) used a novel trigonometric higher-order theory that considers the stretching effect to develop an analytical solution for the static analysis of functionally graded plates. Lee *et al*. (2015) provided an improved higher order shear and normal deformation theory for E, P, and S-FGM plates on Pasternak elastic foundation. Using a new hyperbolic shape function, Akavci and Tanrikulu (2015) provided 2D and quasi-3D shear deformation theories for bending and free vibration analysis of single-layer FG plates. For the bending and free vibration analysis of functionally graded plates resting on elastic foundation, Mantari and Granados (2016) developed an original first shear deformation theory. To examine the static behavior of functionally graded single and sandwich beams with both shear deformation and stretching thickness effects, Yarasca *et al*. (2016) developed a Hermite-Lagrangian finite element formulation. Meftah *et al*. (2017) have conducted a free vibration analysis of FG plates on elastic foundations using a non-polynomial four refined shear deformation theory. To investigate the free vibration of functionally graded plates, Zaoui *et al*. (2017a) and Guerroudj *et al*. (2017) introduced hybrid quasi-3D shear deformation theories. Amar *et al*. (2018) developed a new, straightforward shear deformation theory with three unknowns for the static analysis of FG plates on elastic foundations. The bending, free vibration, and buckling behaviour of FG plates were investigated by Vu *et al*. (2019) utilizing a mesh-free technique and the inverse sin shear deformation plate theory. Levy type porous FGM plates' free vibration and bending analysis were examined by Demirhan and Taskin (2019).

For the purpose of assessing the bending and free vibrations of FGMs plates with simply supported edges, a novel quasi-3D trigonometric HSDTs with a new displacement field with indeterminate integral variables had been devised (Sidhoum *et al*. 2018, Zaoui *et al*. 2020, 2021, 2022b). Mahmoudi *et al*. (2018) examined the effect of micromechanical models on the free vibration of rectangular FGM plate resting on elastic foundation. Younsi *et al*. (2018) developed a non-polynomial 2D and quasi-3D theory to investigate static and dynamic responses of FG plates. Belkhodja *et al*. (2019) investigated the flexion, free vibrations and buckling of FGMs plate with simply supported edges using a new exponential-trigonometric shear function. For FGM plates with the stretching effect present, Bekhodja *et al*. (2022) introduced a novel quasi-3D and 2D hybrid (polynomial-hyperbolic-exponential) HSDT with five unknowns of simply supported square or rectangular plates to study bending, free vibration, and buckling.

The main purpose of the present work is to implement a new 2D and quasi-3D shear strain theories to study the bending behaviour of FG plates simply supported and resting on elastic foundations of the Winkler-Pasternak type. The most intriguing aspect of this theory is that, in contrast to other theories like Neves *et al*. (2012a, 2012b), It has a new displacement field with fewer unknowns. In the proposed field of displacements, transverse shear and thickness stretching effects are taken into account in quasi-3D theory but ignored in the 2D model. Furthermore, in order to satisfy shear stress free boundary conditions without incorporating a shear correction factor, these theories take into account a parabolic variation of transverse shear stresses across the thickness. The fundamental governing equations for FG plates subjected to sinusoidal and uniformly distributed loads are established using the virtual work principle. Navier's method is used to generate closed-form deflection solutions for simply supported plates and the obtained results are then compared to those found in the literature to show the suggested theories' precision and clarity. The bending response of FG plates has been investigated in relation to the impacts of power-law index, slenderness ratio, sideto-thickness ratio, normal strain, Winkler-Pasternak parameters, various rules of mixture and boundary supports.

2. Analytical formulation

2.1. Material properties

In this study, a functionally graded rectangular plate with uniform thickness (h) , length (a) , width (b) , is considered as shown in Fig. 1. The FGM material properties are supposed to change constantly according the plate thickness using the following rules of mixture

2.1.1 The power-law (P-FGM) variation

The volume fraction of the P-FGM plate changes conti-nuously across the thickness of the plate according to the power law variation (Zaoui et al., 2017b) as given in Eq. (1)

$$
P(z) = P_m + (P_c - P_m) \left(\frac{1}{2} + \frac{z}{h}\right)^k
$$
 (1)

2.1.2 The exponential (E-FGM) variation

The E-FGM plate volume fraction vary continuously along the thickness direction of the plate according to the exponential distribution (Meradjah et al., 2018) as shown below

$$
P(z) = Ae^{p(z+h/2)}, \ \ A = P_m, \ \ p = \frac{1}{h} \ln\left(\frac{P_c}{P_m}\right) \tag{2}
$$

2.1.3 The Mori Tanaka Homogenization Model

For Mori–Tanaka scheme (Mori and Tanaka, 1973; Akavci and Tanrikulu, 2015), the volume fraction of the FGM plate is given in the following equation

$$
P(z) = P_m + (P_c - P_m) \frac{V_c}{1 + V_m \left(\frac{P_c}{P_m} - 1\right) \frac{1 + \nu}{3 - 3\nu}},
$$
\n(3a)

with

$$
V_c = \left(\frac{1}{2} + \frac{z}{h}\right)^k, \ \ V_m + V_c = 1 \tag{3b}
$$

where P represents the effective material property like Young's modulus (E) . The properties of the upper and bottom faces of the plate are designated by the letters P_m and P_c , respectively. k is the power law index. Poisson's ratio (V) is assumed constant for all gradation models because its impact on the response of FG plates is relatively minimal.

2.2.Kinematics

Based on higher order shear deformation theory and considering the stretching effect, a novel displacement field of plates is developed as expressed below

$$
u(x, y, z) = u_0(x, y) - z \frac{\partial w_b}{\partial x} - f(z) \frac{\partial^3 w_s}{\partial x \partial y^2}
$$

$$
v(x, y, z) = v_0(x, y) - z \frac{\partial w_b}{\partial y} - f(z) \frac{\partial^3 w_s}{\partial x^2 \partial y}
$$

$$
w(x, y, z) = w_b(x, y) + w_s(x, y) + g(z) \phi_z(x, y)
$$
 (4)

where u_0 , v_0 , w_b , w_s and φ_z are the five unknown displacement functions of mid plate surface. The form function $f(z)$ depicts how the transverse shear strains vary along the thickness. Note that $g(z) = 0$ for 2D problem.

The shape function utilized herein, is presented by Zaoui *et al*. (2019, 2022b) as

$$
f(z) = \frac{\pi h}{\pi^4 + h^4} e^{(hz/\pi)} \left(\pi^2 \sin\left(\frac{\pi z}{h}\right) + h^2 \cos\left(\frac{\pi z}{h}\right) \right) - \frac{\pi h^3}{\pi^4 + h^4}
$$
 (5a)

and
$$
g(z) = \frac{df}{dz}
$$
 (5b)

The general strain – displacement relations can be defined from Eqs. (4) by the application of the linear, small-strain elasticity theory as follows

$$
\begin{Bmatrix} \varepsilon_x \\ \varepsilon_y \\ \gamma_{xy} \end{Bmatrix} = \begin{Bmatrix} \varepsilon_x^0 \\ \varepsilon_y^0 \\ \gamma_{xy}^0 \end{Bmatrix} + z \begin{Bmatrix} k_x^b \\ k_y^b \\ k_y^b \end{Bmatrix} + f(z) \begin{Bmatrix} k_x^s \\ k_y^s \\ k_y^s \end{Bmatrix},
$$
\n(6a)

$$
\begin{Bmatrix} \gamma_{yz} \\ \gamma_{xz} \end{Bmatrix} = \begin{Bmatrix} \gamma_{yz}^0 \\ \gamma_{xz}^0 \end{Bmatrix} + g(z) \begin{Bmatrix} \gamma_{yz}^1 \\ \gamma_{xz}^1 \end{Bmatrix}, \varepsilon_z = g'(z) \varepsilon_z^0
$$
 (6b)

where

$$
\begin{bmatrix} \varepsilon_{x}^{0} \\ \varepsilon_{y}^{0} \\ \gamma_{xy}^{0} \end{bmatrix} = \begin{bmatrix} \frac{\partial u_{0}}{\partial x} \\ \frac{\partial v_{0}}{\partial x} \\ \frac{\partial u_{0}}{\partial y} + \frac{\partial v_{0}}{\partial x} \end{bmatrix}, \quad \begin{bmatrix} k_{x}^{b} \\ k_{y}^{b} \\ k_{xy}^{b} \end{bmatrix} = \begin{bmatrix} -\frac{\partial^{2} w_{b}}{\partial x^{2}} \\ -\frac{\partial^{2} w_{b}}{\partial y^{2}} \\ -2\frac{\partial^{2} w_{b}}{\partial x \partial y} \end{bmatrix}, \tag{7a}
$$

$$
\begin{bmatrix} k_x^s \\ k_y^s \\ k_{xy}^s \end{bmatrix} = \begin{bmatrix} -\frac{\partial w_s^4}{\partial x^2 \partial y^2} \\ -\frac{\partial w_s^4}{\partial x^2 \partial y^2} \\ -\frac{\partial w_s^4}{\partial x \partial y^3} - \frac{\partial w_s^4}{\partial x^3 \partial y} \end{bmatrix},
$$
(7b)

$$
\begin{Bmatrix} \gamma_{yz}^0 \\ \gamma_{xz}^0 \end{Bmatrix} = \begin{Bmatrix} \frac{\partial w_s}{\partial y} \\ \frac{\partial w_s}{\partial x} \end{Bmatrix}, \qquad \begin{Bmatrix} \gamma_{yz}^1 \\ \gamma_{xz}^1 \end{Bmatrix} = \begin{Bmatrix} -\frac{\partial w_s^3}{\partial x^2 \partial y} + \frac{\partial \phi_z}{\partial y} \\ -\frac{\partial w_s^3}{\partial x \partial y^2} + \frac{\partial \phi_z}{\partial x} \end{Bmatrix},
$$
(7c)

$$
\varepsilon_z^0 = \varphi_z \text{ and } g'(z) = \frac{dg(z)}{dz} \tag{7d}
$$

It can be seen from Eq. (6) that the transverse shear strains (γ_{xz} , γ_{yz}) are equal to zero at the top $(z = h/2)$ and bottom $(z = -h/2)$ surfaces of the plate. A shear correction coefficient is, hence, not required.

According to the three-dimensional (3D) elasticity, the stress-strain relationships for FG plates can be expressed as

$$
\begin{bmatrix}\n\sigma_x \\
\sigma_y \\
\sigma_z \\
\sigma_z \\
\tau_{xz} \\
\tau_{xy}\n\end{bmatrix} =\n\begin{bmatrix}\nQ_{11} & Q_{12} & Q_{13} & 0 & 0 & 0 \\
Q_{12} & Q_{22} & Q_{23} & 0 & 0 & 0 \\
Q_{13} & Q_{23} & Q_{33} & 0 & 0 & 0 \\
0 & 0 & 0 & Q_{44} & 0 & 0 \\
0 & 0 & 0 & 0 & Q_{55} & 0 \\
0 & 0 & 0 & 0 & 0 & Q_{66}\n\end{bmatrix}\n\begin{bmatrix}\n\varepsilon_x \\
\varepsilon_y \\
\varepsilon_z \\
\gamma_x \\
\gamma_x \\
\gamma_x \\
\gamma_x \\
\gamma_y\n\end{bmatrix}
$$
\n(8)

The Q_{ij} expressions in terms of engineering constants are depends on the normal strain ε_z .

• In the case of quasi-3D HSDT's, $\varepsilon_z \neq 0$ then Q_{ij} are

$$
Q_{11} = Q_{22} = Q_{33} = \frac{(1 - \nu)E(z)}{(1 - 2\nu)(1 + \nu)}\tag{9a}
$$

$$
Q_{12} = Q_{13} = Q_{23} = \frac{V E(z)}{(1 - 2V)(1 + V)}
$$
(9b)

$$
Q_{44} = Q_{55} = Q_{66} = \frac{E(z)}{2(1+\nu)}\tag{9c}
$$

If the 2D HSDT is used, $\varepsilon_z = 0$ then Q_{ij} are

$$
Q_{11} = Q_{22} = \frac{E(z)}{(1 - v^2)}
$$
 (10a)

$$
Q_{12} = \frac{V E(z)}{(1 - v^2)}
$$
 (10b)

$$
Q_{44} = Q_{55} = Q_{66} = \frac{E(z)}{2(1+\nu)}\tag{10c}
$$

For FG plates, the constitutive relations (8) can be used to precisely calculate the in-plane normal and shear stresses (σ_x , σ_y and τ_{xy}). However, the boundary conditions at the upper and lower plate surfaces may notbe respected if the transverse normal and shear stresses (σ_z , τ_{yz} and τ_{xz}) derived from these constitutive relations. Hence, these stresses are calculated by integrating the 3D elasticity equilibrium equations with respect to the thickness coordinate as

$$
\tau_{xz} = -\int_{-h/2}^{z} \left(\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} \right) dz + C_1(x, y)
$$
\n(11a)

$$
\tau_{yz} = -\int_{-h/2}^{z} \left(\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} \right) dz + C_2(x, y)
$$
\n(11b)

$$
\sigma_{z} = -\int_{-h/2}^{z} \left(\int_{-h/2}^{z} \left[\frac{\partial^{2} \sigma_{x}}{\partial x^{2}} + 2 \frac{\partial^{2} \tau_{xy}}{\partial x \partial y} + \frac{\partial^{2} \sigma_{y}}{\partial y^{2}} \right] dz \right) dz + C_{3}(x, y) z + C_{4}(x, y)
$$
\n(11c)

where C's $(i=1, 4)$ are constants that are specified by the upper and lower boundary conditions of the plate, which are expressed as follows.

$$
\tau_{xz}\big|_{z=\pm h/2} = 0, \; \tau_{yz}\big|_{z=\pm h/2} = 0, \; \sigma_z\big|_{z=h/2} = q(x, y), \sigma_z\big|_{z=-h/2} = 0 \tag{12}
$$

2.3. Equilibrium equations and stress components

By using the concept of virtual works, it is possible to construct the equilibrium equations of functionally graded plate subjected to mechanical applied loads, which can be written analytically as

$$
\int_{-h/2}^{h/2} \int_{A} \left(\sigma_x \delta \varepsilon_x + \sigma_y \delta \varepsilon_y + \sigma_z \delta \varepsilon_z + \tau_{xy} \delta \gamma_{xy} + \tau_{yz} \delta \gamma_{yz} + \tau_{xz} \delta \gamma_{xz} \right) dA dz
$$
\n
$$
- \int_{A} \left(q - f_e \right) \delta w \, dA dz = 0
$$
\n(13)

Where A is the top surface, q is the external load applied to the plate and f_e is the density of foundation reaction effort. For the Pasternak foundation model;

$$
f_e = K_w w - K_{sx} \frac{\partial^2 w}{\partial x^2} - K_{sy} \frac{\partial^2 w}{\partial y^2}
$$
 (14)

where K_w is the transverse elastic coefficient of the foundation and K_{xx} , K_{sy} are coefficients of the shear layer foundation stiffness. If the foundation is homogeneous and isotropic, this means that $K_{xx} = K_{yy} = K_s$. The Winkler foundation replaces the Pasternak foundation if the stiffness of the shear layer foundation is ignored.

By replacing Eqs. (6) and (8) in Eq. (13) and integrating over the thickness of the plate, Eq. (13) can be expressed as

$$
\int_{A} \left[N_x \delta \varepsilon_x^0 + N_y \delta \varepsilon_y^0 + N_z \delta \varepsilon_z^0 + N_{xy} \delta \gamma_{xy}^0 + M_x^b \delta k_x^b + M_y^b \delta k_y^b + M_{xy}^b \delta k_{xy}^b \right. \n+ M_x^s \delta k_x^s + M_y^s \delta k_y^s + M_{xy}^s \delta k_{xy}^s + M_{xz} \delta \gamma_{yz}^0 + M_{yz} \delta \gamma_{xz}^0 + Q_{yz}^s \delta \gamma_{xz}^1 + Q_{xz}^s \delta \gamma_{yz}^1 \right] dA \qquad (15)
$$
\n
$$
- \int_{A} (q - f_e) (\delta w_b + \delta w_s) dA = 0
$$

The stress resultants N, M and Q are given by

$$
\left(N_i, M_i^b, M_i^s\right) = \int_{-h/2}^{h/2} (1, z, f)\sigma_i dz, \ (i = x, y, xy)
$$
\n(16a)

$$
N_z = \int_{-h/2}^{h/2} g'(z) \sigma_z dz
$$
 (16b)

$$
\left(Q_{xz}^s, Q_{yz}^s\right) = \int_{-h/2}^{h/2} g\left(\tau_{xz}, \tau_{yz}\right) dz \tag{16c}
$$

By substituting Eqs. (7) into Eq. (13), integrating the displacement terms in parts and putting the coefficients δu_0 , δv_0 , δw_b , δw_s and $\delta \varphi_z$ to zero, independently, the governing equations of equilibrium can be constructed as follows

$$
\delta u_0 : \frac{\partial N_x}{\partial x} + \frac{\partial N_{xy}}{\partial y} = 0 \tag{17a}
$$

$$
\delta v_0 : \frac{\partial N_y}{\partial y} + \frac{\partial N_{xy}}{\partial x} = 0
$$
\n(17b)

$$
\delta w_b : \frac{\partial^2 M_x^b}{\partial x^2} + \frac{\partial^2 M_y^b}{\partial y^2} + 2 \frac{\partial^2 M_{xy}^b}{\partial x \partial y} - f_e + q = 0 \tag{17c}
$$

$$
\delta w_s : \frac{\partial^4 M_s^s}{\partial x^2 \partial y^2} + \frac{\partial^4 M_s^s}{\partial x^2 \partial y^2} + \frac{\partial^4 M_{xy}^s}{\partial x \partial y^3} + \frac{\partial^4 M_{xy}^s}{\partial x^3 \partial y} + \frac{\partial M_{xz}}{\partial x} + \frac{\partial M_{yz}}{\partial y} - \frac{\partial^3 Q_{xz}^s}{\partial x^2 \partial y} - \frac{\partial Q_{yz}^s}{\partial x \partial y^2} = 0 \tag{17d}
$$

$$
\delta \varphi_z \; : \; \frac{\partial S_{xz}^s}{\partial x} + \frac{\partial S_{yz}^s}{\partial y} - N_z = 0 \tag{17e}
$$

The stresses and moment resultants given in Eqs. (15) can be formulated in terms of generalized displacements (δu_0 , δv_0 , δw_b , δw_s and $\delta \varphi_z$) by substituting Eq. (5) into Eq. (16) and integrating through the thickness of the plate, as

 0 0 0 0 13 23 13 23 13 23 33 66 66 66 12 22 12 22 12 22 23 11 12 11 12 11 12 13 66 11 66 12 22 12 22 12 22 23 11 12 11 12 11 12 13 66 66 66 12 22 12 22 12 22 23 11 12 11 12 11 12 13 0 *z s xy s y s x b xy b y b x xy y x s z xy s y s x b xy b y b x xy y x k k k k k k X X Y Y Y Y Z B D H B B D D H H Y B B D D H H Y B D D B B D D D D Y B B D D D D Y A B B A A B B B B X A A B B B B X N M M M M M M N N N* (18a)

$$
\begin{Bmatrix}\nM_{yz} \\
M_{xz} \\
Q_{yz}^s \\
Q_{xz}^s\n\end{Bmatrix} = \begin{bmatrix}\nA_{44} & 0 & A_{44}^s & 0 \\
0 & A_{55} & 0 & A_{55}^s \\
A_{44}^s & 0 & A_{44}^s & 0 \\
0 & A_{55}^s & 0 & A_{55}^{st}\n\end{bmatrix} \begin{bmatrix}\n\gamma_{yz}^0 \\
\gamma_{xz}^0 \\
\gamma_{yz}^1 \\
\gamma_{yz}^1 \\
\gamma_{xz}^1\n\end{bmatrix}
$$
\n(18b)

where the stiffness components and inertias are given as

$$
(A_{ij}, A_{ij}^s, B_{ij}, D_{ij}, B_{ij}^s, D_{ij}^s, H_{ij}^s) = \int_{-h/2}^{h/2} Q_{ij} (1, g^2(z), z, z^2, f(z), z f(z), f^2(z)) dz
$$
 (19a)

$$
(X_{ij}, Y_{ij}, Y_{ij}^s, Z_{ij}) = \int_{-h/2}^{h/2} (1, z, f(z), g'(z)) g'(z) Q_{ij} dz
$$
 (19b)

3. Navier-type solutions for simply supported FG plates

The Navier's procedure (Shao and Ma, 2007; Grover et al., 2013), based on double Fourier series, is employed herein to define the closed-form solution of the partial differential equations (Eqs.(18)) for which the displacement variables satisfying the boundary conditions can be given as

$$
\begin{Bmatrix}\nu_0 \\
v_0 \\
w_b \\
w_s \\
w_s\n\end{Bmatrix} = \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} \sum_{n=1}^{\infty} \begin{Bmatrix}\nU_{mn} \cos(\lambda x) \sin(\mu y) \\
V_{mn} \sin(\lambda x) \cos(\mu y) \\
W_{bmn} \sin(\lambda x) \sin(\mu y) \\
W_{sm} \sin(\lambda x) \sin(\mu y) \\
\Phi_{mn} \sin(\lambda x) \sin(\mu y)\n\end{Bmatrix}
$$
\n(20)

where $(U_{mn}, V_{mn}, W_{bm}, W_{mn}, \Phi_{mn})$ are unknown functions to be determined. λ and μ are expressed as

$$
\lambda = m\pi / a \text{ and } \mu = n\pi / b \tag{21}
$$

The transverse distributed load $q(x, y)$ is also expanded in a double-Fourier series as

$$
q(x, y) = \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} q_{mn} \sin\left(\frac{m\pi}{a}x\right) \sin\left(\frac{n\pi}{b}y\right)
$$
 (22)

The coefficients q_{mn} are given below for some general loadings

For uniformly distributed load

$$
q_{mn} = \begin{cases} \frac{16q_0}{mn\pi^2}, & m, n = 1, 3, 5, \dots \\ 0 & m, n = 2, 4, 6, \dots \end{cases}
$$
 (23)

For sinusoidal distributed load

$$
q_{mn} = q_0 \tag{24}
$$

in which q_0 is the intensity of the load.

In order to obtain closed-form solutions (Eq. (25)) for the static issue of the FG plate, Eqs. (18), (20) and (22) must be substituted into Eq. (17).

$$
\begin{bmatrix}\nS_{11} & S_{12} & S_{13} & S_{14} & S_{15} \\
S_{12} & S_{22} & S_{23} & S_{24} & S_{25} \\
S_{13} & S_{23} & S_{33} & S_{34} & S_{35} \\
S_{14} & S_{24} & S_{34} & S_{44} & S_{45} \\
S_{15} & S_{25} & S_{35} & S_{45} & S_{55}\n\end{bmatrix}\n\begin{bmatrix}\nU_{mn} \\
V_{mn} \\
W_{bm} \\
W_{mm} \\
W_{mm} \\
W_{mm}\n\end{bmatrix} =\n\begin{bmatrix}\n0 \\
0 \\
q_{mn} \\
q_{mn} \\
0\n\end{bmatrix}
$$
\n(25)

in which

$$
S_{11} = \lambda^2 A_{11} + \mu^2 A_{66}, S_{12} = \lambda \mu (A_{12} + A_{66})
$$

\n
$$
S_{13} = -\lambda^3 B_{11} - \lambda \mu^2 (B_{12} + 2B_{66})
$$

\n
$$
S_{14} = \lambda^3 \mu^2 (B_{11}^s + B_{12}^s) + (\lambda \mu^4 + \lambda^3 \mu^2) B_{66}^s
$$

\n
$$
S_{15} = -\lambda X_{13}
$$
\n(26)

$$
S_{22} = \lambda^2 A_{66} + \mu^2 A_{22}
$$

\n
$$
S_{23} = -\mu^3 B_{22} - \lambda^2 \mu (B_{12} + 2B_{66})
$$

\n
$$
S_{24} = \lambda^2 \mu^3 (B_{12}^s + B_{22}^s) + (\lambda^2 \mu^3 + \lambda^4 \mu) B_{66}^s
$$

\n
$$
S_{25} = -\mu X_{23}
$$

\n
$$
S_{33} = \lambda^4 D_{11} + \mu^4 D_{22} + 2\lambda^2 \mu^2 (D_{12} + 2D_{66})
$$

\n
$$
+ K_w + \lambda^2 K_{sx} + \mu^2 K_{sy}
$$

\n
$$
S_{34} = -\lambda^2 \mu^2 (\lambda^2 D_{11}^s + \mu^2 D_{22}^s) + (\lambda^2 \mu^4 + \lambda^4 \mu^2)
$$

\n
$$
(D_{12}^s + 2D_{66}^s) - 2\alpha^2 \beta^2 (k_1 A' + k_2 B') D_{66}^s
$$

\n
$$
S_{35} = \lambda^2 Y_{13} + \mu^2 Y_{23}
$$

\n
$$
S_{44} = \lambda^4 \mu^4 (H_{11}^s + 2H_{12}^s + H_{22}^s + 2H_{66}^s) + \lambda^2 \mu^2 (\lambda^4 + \mu^4) H_{66}^s
$$

\n
$$
+ 2\lambda^2 \mu^2 (As_{44} + As_{55}) + \lambda^2 \mu^2 (\mu^2 As_{55} + \lambda^2 As_{44})
$$

\n
$$
S_{45} = \lambda^2 \mu^2 (As_{455}^s + As_{44}^s - Y_{13}^s - Y_{23}^s) + \lambda^2 A_{55}^s + \mu^2 A_{44}^s
$$

\n
$$
S_{55} = \lambda^2 A_{55}^s + \mu^2 A_{44}^s + Z_{33}
$$

4.Results and Discussion

4.1. Bending study of FG plates with simple supports

The accuracy of the current theory in analyzing thebending behavior of simply supported FG plates is examined in this section through the presentation of several numerical examples and comparison with the findings of various 2D, 3D, and quasi-3D shear deformation theories. Table 1 lists the mechanical properties of the metal and ceramic materials employed in this study. Both homogeneous isotropic plates and FGPs are investigated in the computations, and parameters study were also establshed. The numerical findings are presented in graphical and tabular forms by using the following nondimensional displacements and stresses

$$
\overline{w} = \frac{10h^3 E_c}{a^4 q_0} w\left(\frac{a}{2}, \frac{b}{2}, z\right), \overline{\sigma}_x = \frac{h}{aq_0} \sigma_x \left(\frac{a}{2}, \frac{b}{2}, z\right), \overline{\sigma}_y = \frac{h}{aq_0} \sigma_y \left(\frac{a}{2}, \frac{b}{2}, z\right),
$$
\n
$$
\overline{\tau}_{xy} = \frac{h}{aq_0} \tau_{xy} (0, 0, z), \overline{\tau}_{xz} = \frac{h}{aq_0} \tau_{xz} \left(0, \frac{b}{2}, z\right), \overline{\tau}_{yz} = \frac{h}{aq_0} \tau_{yz} \left(\frac{a}{2}, 0, z\right)
$$
\n
$$
\widetilde{u} = \frac{E_c}{hq_0} u\left(0, \frac{b}{2}, z\right), \widetilde{w} = \frac{E_c}{hq_0} w\left(\frac{a}{2}, \frac{b}{2}, z\right), \widetilde{\sigma}_x = \frac{\sigma_x \left(\frac{a}{2}, \frac{b}{2}, z\right)}{q_0},
$$
\n
$$
\widetilde{\sigma}_z = \frac{\sigma_z \left(\frac{a}{2}, \frac{b}{2}, z\right)}{q_0}, \widetilde{\tau}_{xy} = \frac{\tau_{xy} (0, 0, z)}{q_0}, \widetilde{\tau}_{xz} = \frac{\tau_{xz} \left(0, \frac{b}{2}, z\right)}{q_0}
$$
\n
$$
\hat{u} = \frac{G_1}{hq_0} u\left(0, \frac{b}{2}, z\right), \hat{w} = \frac{G_1}{dq_0} w\left(\frac{a}{2}, \frac{b}{2}, z\right),
$$
\n(27)

4.1.1. Functionally graded plates (P-FGM)

In this section, non-dimensional displacements and stresses of an Al / Al_2O_3 FG square plate under uniformly and sinusoidal distributed loads for various values of the power-law index are calculated and presented in Tables 2 and 3.

In Table 2, the computed results of non-dimensional deflection and stress components of moderately thick square FG plate under uniform load are compared with those given by quasi-3D and 2D shear deformation theories by Akavci and Tanrikulu (2015) and Younsi et al. (2018). From this table, it can be observed that the results of the proposed 2D theory agree well with those of Akavci and Tanrikulu (2015) and Younsi et al. (2018) in all cases. Besides, by comparing the obtained values by the present quasi-3D theory with those given by the other quasi-3D theories, an excellent correlation can be remarked.

Table 3presents the non-dimensional normal stress $(\overline{\sigma}_x)$ and transverse displacement (\overline{w}) of square thin and thick plates subjected to sinusoidal load forthree different power law index (k). The predicted results are compared with those generated by Carerra et al. (2011) based on a fourth-order variations of both in-plane and transverse displacements across the thickness, quasi-3D sinusoidal and hyperbolic shear deformation theories of Neves et al. (2012a, 2012b), quasi-3D shear deformation theories of Hebali (2014), 2D and quasi-3D hyperbolic shear deformation theories of Akavci and Tanrikulu (2015) and Younsi et al. (2018). A good agreement between the results is found for both thin and thick plates. Also, the present model provides a good prediction of both displacement and stress even in thick FG plates where the stretching effects are more pronounced.

Moreover, it should be noticed that the number of unknown variables used in the present formulation is just four in the 2D theory and five in the quasi-3D model, while nine unknown functions were used in hyperbolic shear deformation theories of Neves et al. (2012a, 2012b). It can be concluded that the present theory is not only accurate but also comparatively simple in predicting the bending response of simply supported FGM plates.

The stresses and displacements variations through the thickness of $A1/A1_2O_3$ FG square plate subjected to sinusoidal load are plotted in Fig. 2. The results are shown as compared with the quasi-3D shear deformation theory of Akavci and Tanrikulu (2015) for different values of power-law index *^k* . From this figure, it can be observedan excellent agreement between the obtained results and those computed by Akavci and Tanrikulu (2015). It is important to stress out that, the in-plane stresses $(\overline{\sigma}_x)$ and $(\bar{\tau}_{xy})$ exhibit linear variation through the thickness of homogeneous plate while it is parabolic for FG plates. This figure also shows that, the deflection (\overline{w}) and in-plane stresses $(\overline{\sigma}_x)$ and $(\overline{\sigma}_y)$ increase and the shear stresses $(\bar{\tau}_{xz})$ and $(\bar{\tau}_{xy})$ diminish with the increasing value of material index (k)

Fig. 3 presents a 3D interaction diagram of the power-law index (k) , side-to-thickness ratio (a/h) and center deflection (\overline{w}) using the proposed 2D and quasi-3D theories. It can be seen from this figure that the center deflection increases with the increase of the power-law index and decreases with the increasing of the thickness ratio. It is due to the fact that a higher value of (k) corresponds to lower value of volume fraction of the ceramic phase, and thus leads to the decrease of the value of the elasticity modulus which makes the plate softer. It also can be shown that the center deflections computed from the present 2D theory which neglect the thickness stretching effect are higher than those calculated from quasi-3D theory.

4.1.2. Exponentially graded plates (E-FGM)

.

Computations in this section are carried out for a simply supported E-FGM plate. The material properties of the E-FGM plate are defined by the exponential function given in Eq. (2). The nondimensional stress and displacements of the E-FGM plate are calculated and compared with the results of different HSDTs for different loadings.

The center deflections (\overline{w}) , in-plane and transverse shear stresses $(\overline{\sigma}_x, \overline{\tau}_x)$ of Al/Al₂O₃ plates under sinusoidal loads are calculated for different values of aspect ratio (b/a) , thickness ratio (a/h) and exponent values (k) in Tables 4-6. The central deflections of the very thick E-FGM plates are analyzed in Table 4. The obtained predictions are compared with the quasi-3D sinusoidal and exact 3D elasticity theories of Zenkour (2007), 2D and quasi-3D trigonometric models of Mantari and Soares (2013) and the quasi-3D and 2D shear deformation theories by Akavci and Tanrikulu (2015), Younsi et al. (2018). Since the proposed and other quasi-3D models include the thickness-stretching influence, they lead to results close to each other. Whereas, 2D HSDTs overestimate the deflections due to omitting the thickness stretching influence. In Tables 5 and 6, the computed non-dimensional stresses are provided as compared with those given by Younsi et al. (2018), Akavci and Tanrikulu (2015) based on quasi-3D and 2D hyperbolic theories and Mantari and Soares (2013) using 2D and quasi-3D trigonometric theories. It can be seen from the table that an excellent agreementis achieved between the results of present theory and those of other theories. Tables 4–6 demonstrate also that deflection (\overline{w}) and transverse shear stress (\overline{z}_x) diminish and in plane stress $(\overline{\sigma}_x)$ increases with the increase of the exponent (k) .

In Fig. 4, the variation of non-dimensional displacements and stresses according to the thickness of an E-FGM plate subjected to sinusoidal loading for different E_0/E_1 ratios are displayed using the present theory with including the thickness stretching effect and compared with the quasi-3D theory of Younsi et al. (2018). A very good accuracy between the solutions is observed. It can be seen also from these results that the non-dimensional displacements increase with the increasing of E_0/E_1 . In addition, it can be deduced that E_0/E_1 ratios affect considerably the non-dimensional stresses.

4.1.3. Comparative study

In this party, parametric studies have been presented to evaluate the effect of power law index (k) and side-to-thickness ratio (a/h) on the bending of functionally graded plates using three rules of mixture (P-FGM, E-FGM, Mori-Tanaka homogenization model). Fig. 5 illustrates the variation of non-dimensional deflection (\overline{w}) with respect to exponent index (k) and slenderness ratio (a/h) for P-FGM, E-FGM, Mori-Tanaka homogenization scheme. It can be noted from this figure, that deflections of plates using E-FGM decrease and by using P-FGM and Mori-Tanaka homogenization model increase when the exponent index increases. Such behavior is due to the fact that the increase of power-law index will increase the stiffness of the EGM plate, and thus, lead to a reduction of transverse displacement.

Fig. 6 is devoted to present the influence of side-to-thickness ratio on non-dimensional deflection of simply supported plate with different models. As can be observed from this figure, increasing of side-to-thickness ratio causes reducing of the magnitude of deflection, it means that the effect of shear deformation is significant when plates are thick $(a/h \leq 5)$ and negligible for thin plates. Further, it is apparent that plates made with exponential function have the lowest deflection than the other functions.

Another comparative study for evaluating the dimensionless center deflections of Mori-Tanaka homogenization scheme, P-FGM and E-FGM plates subjected to sinusoidal load is carried out. From Fig. 7, it is showed that the deflection at the center of plates with Mori-Tanaka homogenization model is larger than those of P-FGM and E-FGM plates. The E-FGM plate which has the smallest deflection is stiffer than the other FGM plates.

4.2. Analysis of FG plates on elastic foundation

This section aims to demonstrate the accuracy of the proposed new models in predicting the static response of FG plates resting on elastic foundation.The following relations of non-dimensional displacements, stresses and foundation parameters in the bending problem are used

$$
u^* = \frac{100D_c}{q_0 a^4} u\left(0, \frac{b}{2}, -\frac{h}{2}\right), v^* = \frac{100D_c}{q_0 a^4} v\left(\frac{b}{2}, 0, -\frac{h}{2}\right),
$$

$$
w^* = \frac{100D_c}{q_0 a^4} w\left(\frac{a}{2}, \frac{b}{2}\right), \sigma_x^* = -\frac{h^2}{q_0 a^2} \sigma_x \left(\frac{a}{2}, \frac{b}{2}, -\frac{h}{2}\right)
$$

$$
\sigma_{y}^* = -\frac{h^2}{q_0 a^2} \sigma_y \left(\frac{a}{2}, \frac{b}{2}, -\frac{h}{2}\right), \sigma_{xy}^* = \frac{h^2}{q_0 a^2} \sigma_{xy} \left(0, 0, -\frac{h}{2}\right)
$$
\n
$$
K_0 = \frac{K_w a^4}{E_0 h^3}, J_0 = \frac{K_s a^2}{v E_0 h^3}, E_0 = 1 GPa
$$
\n
$$
\overline{K}_w = \frac{K_w a^4}{D_c}, \overline{K}_s = \frac{K_s a^2}{D_c}, D_c = \frac{E_c h^3}{12(1 - v^2)}
$$
\n(28)

In the first example, isotropic square plates on elastic foundation are analyzed to verify the accuracy of the proposed theories. The dimensionless deflection of homogeneous square plate subjected to uniform loads is presented in Table 8 for two values of the side-to-thickness ratio (a/h) and different values of the foundation parameters $(\overline{K}_{w}, \overline{K}_{s})$. The obtained results are compared with those given by Thai and Choi (2014a) using differential quadrature method and those of Al Khateeb and Zenkour (2014) based on refined shear deformation plate theory. This table proves that the computed results are in excellent agreement with those reported by the other theories of Thai and Choi (2014a), Al Khateeb and Zenkour (2014) for all values of side-to-thickness ratio and foundation parameters.

The next example is implemented for A/Al_2O_3 moderately thick rectangular plates on elastic foundation. Table 9 presents a comparison of non-dimensional displacements and stresses of FG plate for various exponent value (k) and foundation parameters (K_0, J_0) derived from the present theory, the sinusoidal shear deformation theory of Zenkour (2009), the 2D Zeroth-order shear deformation theory of Thai and Choi (2014a) and the first shear deformation theory of Mantari and Granados (2016). It is explicit that the present results are again found more close in all cases. It should be noted thatthe present theory is not only efficient but more accurate in predicting the bending behavior of FG plates resting on elastic foundation. The table exhibits that, axial displacements, deflection and stresses decreases with the increase of foundation parameters. In addition, it is apparent that the computed values from the present quasi-3D theory which takes into account the thickness stretching effect are smaller than those calculated from 2D theory.

To check the effect of Winkler's and Pasternak's foundation parameters $(\overline{K}_{w}, \overline{K}_{x})$ on the bending behavior of FG plates subjected to sinusoidal loads based on the proposed 2D and quasi-3D theories, Fig. 8 depicts the variation of non-dimensional deflection (\overline{w}) of Al/Al₂O₃ FG square plates versus Winkler (\overline{K}_w) and Pasternak (\overline{K}_s) parameters. As can be seen from this figure, the dimensionless deflection diminishes when the foundation parameters increase. Compared to the Winkler parameter (\overline{K}_w) , the Pasternak foundation parameter (\overline{K}_s) has more significant effect on decreasing the dimensionless deflection. Besides, the deflection computed by present quasi-3D theory are lower than those obtained from 2D theory which means the thickness stretching effect is more significant when the thickness of the plate is higher, but it is negligible with reducing the thickness of the plate. This indicates that the stretching effect is pronounced, and must be taken into account in the modelling of thick plate.

5. Conclusions

This study examines the bending response of FG plates resting on elastic foundations using a new developed 2D and quasi-3D shear strain theories. A new shear shape function that meets the boundary conditions with no tension or compression on the bottom and top surfaces of the plate is used in the theory, which only uses five unknowns, to account for a parabolic distribution of the transverse shear stress. Moreover, the shear correction factor has not been considered. The equilibrium equations were determined using the virtual work principle. In order to show the validity of the proposed theory, the analytical solutions of simply supported plates are produced using the double Fourier series and are compared to previously published solutions. The effects of a large number of parameters such as the

transverse normal strain, the form ratio of the plate, the length/thickness ratio, the volume fraction distributions, the various mixing laws as well as the boundary conditions on the deflection, axial displacements and the various stresses are examined in detail. The above-analyzed results allow for the following key conclusions:

- It is clear from all of the comparative experiments that the proposed theory exhibits strong agreement with the findings of previous 2D and quasi-3D HSDTs.
- Although there are five unknowns in the proposed quasi-3D HSDT, the results it produces are comparable to those of other published quasi-3D theories with more unknowns, such as the quasi-3D theories of Neves et al. (2012a, 2012b), which have nine unknowns, and Akavci and Tanrikulu (2015), which have six unknowns.
- The power law index affects how much a plate deflects. A material index number greater than 5 has very little impact.
- The thickness stretching effect was not taken into account, which resulted in the little disparity between the current 2D and quasi-3D shear deformation values.
- The findings show that the plate stiffens when the effects of normal deformations are taken into account, which reduces deflection and increases stresses.
- For thick plates, the thickness stretching effect is more pronounced, therefore it must be accounted in the modeling.

Finally, it can be concluded that the formulation developed here in can serve as a reference to future research and could be extended to the analysis of classical and thermal buckling of FG plates by introducing various combined boundary conditions.

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Fig. 1 Geometry and coordinates of the considered FGM plate on elastic foundation.

Fig. 2Distributions of the non-dimensional displacements and stresses through the thickness of square P-FGM plate ($a/h = 10$)

Fig. 3 Effect of the power-law index k and side-to-thickness ratio a/h on the non-dimensional deflection \overline{w} of simply supported Al / Al_2O_3 P-FGM plates.

Fig. 4 Distributions of the non-dimensional displacements and stresses through the thickness of square E-FGM plate $(a/h = 10)$

Fig.5 Non-dimensional deflection variation \overline{w} of different models of FG plates for various values of power-law index k and side-to-thickness ratio (a/h) .

Fig.6 Effect of the side-to-thickness ratio (a/h) on the non-dimensional deflection of different FG plates.

Fig. 7 Comparison of the deflection of Mori-Tanaka model, P-FGM and E-FGM plates $(a/h = 4, k = 2).$

Fig.8 Effect of foundation parameters $(\overline{K}_w, \overline{K}_s)$ on the non-dimensional deflection \overline{w} of FG square plates under sinusoidal load $(k = 1, a/h = 10)$.

Table 1 Material properties used in the FG plates.

Material	Properties		
	E(GPa)		(kg/m^3)
Aluminum (Al)		0.3	2702
Alumina $(Al2O3)$	380	0.3	3800
Zirconia $(ZrO2)$	200	0 3	5700

Table 2 The non-dimensional displacement and stress components of an Al/Al₂O₃ FG square plate subjected to uniformly distributed load ($a/h = 10$).

k	Theory	ε_z		$\overline{\sigma}_x(h/3)$				
			$a/h = 4$	$a/h = 10$	$a/h = 100$	$a/h = 4$	$a/h = 10$	$a/h = 100$
$\mathbf{1}$	Carrera et al. (2011)	$\neq 0$	0.6221	1.5064	14.9690	0.7171	0.5875	0.5625
	Neves et al. (2012a)	$\neq 0$	0.5925	1.4945	14.9690	0.6997	0.5845	0.5624
	Neves et al. (2012b)	$\neq 0$	0.5910	1.4917	14.9440	0.7020	0.5868	0.5648
	Hebali et al. (2014)	$\neq 0$	0.5952	1.4954	14.9630	0.6910	0.5686	0.5452
	Akavci and Tanrikulu (2015)	$= 0$	0.5806	1.4895	14.9670	0.7282	0.5889	0.5625
	Akavci and Tanrikulu (2015)	$\neq 0$	0.5754	1.4322	14.3060	0.6908	0.5691	0.5457
	Younsi et al. (2018)	$= 0$	0.5808	1.4896	14.9675	0.7283	0.5889	0.5625
	Younsi et al. (2018)	$\neq 0$	0.5758	1.4330	14.3135	0.6910	0.5692	0.5459
	Present study	$= 0$	0.5803	1.4894	14.9675	0.7280	0.5889	0.5625
	Present study	$\neq 0$	0.5705	1.4157	14.1330	0.6896	0.5680	0.5447
$\overline{4}$	Carrera et al. (2011)	$\neq 0$	0.4877	1.1971	11.9230	1.1585	0.8821	0.8286
	Neves et al. (2012a)	$\neq 0$	0.4404	1.1783	11.9320	1.1178	0.8750	0.8286
	Neves et al. (2012b)	$\neq 0$	0.4340	1.1593	11.7380	1.1095	0.8698	0.8241
	Hebali et al. (2014)	$\neq 0$	0.4507	1.1779	11.8710	1.0964	0.8413	0.7926
	Akavci and Tanrikulu (2015)	$= 0$	0.4431	1.1787	11.9200	1.1613	0.8818	0.8287
	Akavci and Tanrikulu (2015)	$\neq 0$	0.4247	1.1017	11.0880	1.0983	0.8417	0.7925
	Younsi et al. (2018)	$= 0$	0.4437	1.1789	11.9209	1.1609	0.8817	0.8287
	Younsi et al. (2018)	$\neq 0$	0.4260	1.1045	11.1152	1.0982	0.8419	0.7928
	Present study	$= 0$	0.4424	1.1783	11.9208	1.1618	0.8818	0.8287
	Present study	$\neq 0$	0.4181	1.0802	10.8633	1.0970	0.8403	0.7910
10	Carrera et al. (2011)	$\neq 0$	0.3965	0.8965	8.9077	1.3745	1.0072	0.9361
	Neves et al. $(2012a)$	$\neq 0$	0.3227	1.1783	11.9320	1.3490	0.8750	0.8286
	Neves et al. (2012b)	$\neq 0$	0.3108	0.8467	8.6013	1.3327	0.9886	0.9228
	Hebali et al. (2014)	$\neq 0$	0.3325	0.8889	8.9977	1.3333	0.9791	0.9114
	Akavci and Tanrikulu (2015)	$= 0$	0.3242	0.8778	8.9059	1.3917	1.0089	0.9362
	Akavci and Tanrikulu (2015)	$\neq 0$	0.3095	0.8229	8.3185	1.3352	0.9818	0.9141
	Younsi et al. (2018)	$= 0$	0.3248	0.8780	8.9059	1.3915	1.0088	0.9362
	Younsi et al. (2018)	$\neq 0$	0.3109	0.8259	8.3473	1.3353	0.9819	0.9141
	Present study	$= 0$	0.3235	0.8775	8.9059	1.3917	1.0089	0.9362
	Present study	$\neq 0$	0.3033	0.8031	8.1118	1.3333	0.9807	0.9130

Table 3 Non-dimensional displacement and stress of an Al/Al₂O₃ FG square plate subjected to sinusoidal load.

Table 4 Non-dimensional deflection $\bar{w}(0) = \frac{10h^3 E_0}{a^4 q_0}$ $(0) = \frac{10h^3 E_0}{a^4 a_0} w \left(\frac{a}{2}, \frac{b}{2}, 0 \right)$ $\overline{w}(0) = \frac{10h^3E_0}{\overline{w}}$ $w\left(\frac{a}{\overline{w}},\frac{b}{\overline{w}}\right)$ *^a q* $=\frac{10h^3E_0}{a^4a_0}$ $\left(\frac{a}{2},\frac{b}{2},0\right)$ of E-FGM plates subjected to sinusoidal distributed load ($a/h = 2$).

b/a	Theory	ε_z	\boldsymbol{k}					
			0.1	0.3	0.5	0.7	1	1.5
$\mathbf{1}$	Zenkour (2007)	$\neq 0$	0.5769	0.5247	0.4766	0.4324	0.3726	0.2890
	Zenkour (2007)	$= 0$	0.5730	0.5180	0.4678	0.4221	0.3611	0.2771
	Mantari and Soares (2016)	$\neq 0$	0.5778	0.5224	0.4717	0.4256	0.3648	0.2793
	Mantari and Soares (2016)	$= 0$	0.6362	0.5751	0.5194	0.4687	0.4017	0.3079
	Akavci and Tanrikulu (2015)	$= 0$	0.6351	0.5741	0.5185	0.4679	0.4004	0.3075
	Akavci and Tanrikulu (2015)	$\neq 0$	0.5750	0.5198	0.4694	0.4236	0.3624	0.2780
	Younsi et al. (2018)	$= 0$	0.6355	0.5745	0.5189	0.4683	0.4007	0.3077
	Younsi et al. (2018)	$\neq 0$	0.5758	0.5205	0.4701	0.4242	0.3629	0.2784
	Present study	$= 0$	0.6343	0.5734	0.5179	0.4674	0.4000	0.3072
	Present study	$\neq 0$	0.5731	0.5181	0.4679	0.4222	0.3612	0.2771
$\overline{2}$	Zenkour (2007)	$\neq 0$	1.1944	1.0859	0.9864	0.8952	0.7726	0.6017
	Zenkour (2007)	$= 0$	1.1879	1.0739	0.9700	0.8754	0.7493	0.5757
	Mantari and Soares (2016)	$\neq 0$	1.1940	1.0794	0.9750	0.8799	0.7537	0.5786
	Mantari and Soares (2016)	$= 0$	1.2776	1.1553	1.0441	0.9430	0.8092	0.6237
	Akavci and Tanrikulu (2015)	$= 0$	1.2763	1.1541	1.0431	0.9422	0.8079	0.6234
	Akavci and Tanrikulu (2015)	$\neq 0$	1.1938	1.0765	0.9723	0.8775	0.7511	0.5771
	Younsi et al. (2018)	$= 0$	1.2768	1.1546	1.0435	0.9426	0.8082	0.6236
	Younsi et al. (2018)	$\neq 0$	1.1917	1.0774	0.9731	0.8782	0.7517	0.5775
	Present study	$= 0$	1.2753	1.1532	1.0423	0.9415	0.8074	0.6231
	Present study	$\neq 0$	1.1880	1.0740	0.9701	0.8755	0.7494	0.5758
3	Zenkour (2007)	$\neq 0$	1.4429	1.3116	1.9112	1.0811	0.9333	0.7275
	Zenkour (2007)	$= 0$	1.4354	1.2977	1.1722	1.0579	0.9056	0.6961
	Mantari and Soares (2016)	$\neq 0$	1.4421	1.3037	1.1776	1.0627	0.9104	0.6992
	Mantari and Soares (2016)	$= 0$	1.5340	1.3873	1.2540	1.1329	0.9725	0.7506
	Akavci and Tanrikulu (2015)	$= 0$	1.5327	1.3861	1.2530	1.1320	0.9712	0.7503
	Akavci and Tanrikulu (2015)	$\neq 0$	1.4386	1.3005	1.1748	1.0602	0.9076	0.6976
	Younsi et al. (2018)	$= 0$	1.5332	1.3866	1.2534	1.1324	0.9715	0.7504
	Younsi et al. (2018)	$\neq 0$	1.4396	1.3015	1.1756	1.0610	0.9082	0.6981
	Present study	$= 0$	1.5316	1.3852	1.2521	1.1313	0.9706	0.7499
	Present study	$\neq 0$	1.4354	1.2977	1.1722	1.0579	0.9057	0.6961

Table 5 Non-dimensional stress $\bar{\sigma}_{r} \left(\frac{h}{r} \right) = \frac{h^2}{r^2}$ $a^2 q_0$ x^2 (2'2'2 $h \rightarrow h^2$ (a b h *^a q* $\bar{\sigma}_x \left(\frac{h}{2}\right) = \frac{h^2}{a^2 q_0} \sigma_x \left(\frac{a}{2}, \frac{b}{2}, \frac{h}{2}\right)$ of E-FGM plates subjected to sinusoidal

b/a	Theory	\mathcal{E}_z	\boldsymbol{k}								
			0.1	0.3	0.5	0.7	1	1.5	$\overline{2}$	2.5	3
$\mathbf{1}$	Mantari and Soares (2016)	$\neq 0$	0.2196	0.2345	0.2503	0.2671	0.2944	0.3460	0.4065	0.4775	0.5603
	Mantari and Soares (2016)	$= 0$	0.2062	0.2204	0.2355	0.2515	0.2774	0.3264	0.3835	0.4502	0.5278
	Akavci and Tanrikulu (2015)	$= 0$	0.2063	0.2205	0.2356	0.2516	0.2776	0.3266	0.3838	0.4504	0.5281
	Akavci and Tanrikulu (2015)	$\neq 0$	0.2142	0.2285	0.2438	0.2601	0.2866	0.3370	0.3964	0.4664	0.5485
	Younsi et al. (2018)	$= 0$	0.2063	0.2205	0.2355	0.2516	0.2775	0.3265	0.3837	0.4504	0.5279
	Younsi et al. (2018)	$\neq 0$	0.2137	0.2280	0.2433	0.2595	0.2860	0.3363	0.3957	0.4657	0.5478
	Present study	$= 0$	0.2063	0.2205	0.2356	0.2517	0.2776	0.3266	0.3838	0.4505	0.5282
	Present study	$\neq 0$	0.2195	0.2344	0.2502	0.2670	0.2943	0.4359	0.4064	0.4773	0.5602
$\overline{2}$	Mantari and Soares (2016)	$\neq 0$	0.4552	0.4867	0.5200	0.5554	0.6126	0.7201	0.8449	0.9898	1.1580
	Mantari and Soares (2016)	$= 0$	0.4350	0.4649	0.4966	0.5303	0.5850	0.6881	0.8085	0.9490	1.1125
	Akavci and Tanrikulu (2015)	$= 0$	0.4351	0.4650	0.4968	0.5305	0.5852	0.6884	0.8088	0.9493	1.1129
	Akavci and Tanrikulu (2015)	$\neq 0$	0.4466	0.4773	0.5098	0.5443	0.6002	0.7058	0.8289	0.9725	1.1397
	Younsi et al. (2018)	$= 0$	0.4351	0.4650	0.4967	0.5305	0.5851	0.6883	0.8087	0.9492	1.1128
	Younsi et al. (2018)	$\neq 0$	0.4459	0.4765	0.5090	0.5435	0.5993	0.7048	0.8278	0.8278	1.1388
	Present study	$= 0$	0.4351	0.4650	0.4968	0.5306	0.5852	0.6884	0.8089	0.9494	1.1131
	Present study	$\neq 0$	0.4551	0.4865	0.5199	0.5553	0.6124	0.7199	0.8447	0.9897	1.1579
$\overline{3}$	Mantari and Soares (2016)	$\neq 0$	0.5514	0.5896	0.6302	0.6733	0.7427	0.8730	1.0240	1.1990	1.4017
	Mantari and Soares (2016)	$= 0$	0.5288	0.5651	0.6037	0.6447	0.7112	0.8365	0.9828	1.1536	1.3523
	Akavci and Tanrikulu (2015)	$= 0$	0.5290	0.5653	0.6039	0.6449	0.7114	0.8368	0.9832	1.1540	1.3528
	Akavci and Tanrikulu (2015)	$\neq 0$	0.5418	0.5791	0.6187	0.6608	0.7289	0.8570	1.0061	1.1797	1.3813
	Younsi et al. (2018)	$= 0$	0.5289	0.5652	0.6038	0.6449	0.7113	0.8367	0.9831	1.1538	1.3527
	Younsi et al. (2018)	$\neq 0$	0.5410	0.5783	0.6179	0.6599	0.7279	0.8559	1.0050	1.1786	1.3803
	Present study	$= 0$	0.5290	0.5653	0.6039	0.6450	0.7114	0.8368	0.9833	1.1541	1.3529
	Present study	$\neq 0$	0.5512	0.5895	0.6300	0.6731	0.7425	0.8728	1.0238	1.1988	1.4016

distributed load ($a/h = 10$).

Table 6 Non-dimensional stress $\bar{\tau}_{xz}(0) = \frac{h}{aq_0}$ $\tau_{xz}(0) = \frac{1}{a a_0} \tau_{xz} (0, \frac{1}{2}, 0)$ *h b* $\overline{\tau}_{xz}(0) = \frac{h}{aq_0} \tau_{xz} \left(0, \frac{b}{2}, 0\right)$ of E-FGM plates subjected to sinusoidal distributed load ($a/h = 10$).

b/a	Theory	ε .	r								
			0.1	0.3	0.5	0.7				γ	
	Mantari and									0.2454 0.2450 0.2442 0.2430 0.2405 0.2344 0.2263 0.2162 0.2045	
	Soares (2016)										
	Mantari and	$= 0$	0.2380	0.2376	0.2368				0.2356 0.2330 0.2268 0.2185 0.2094		0.1985

	Soares (2016)										
	Akavci and										
	Tanrikulu	$= 0$	0.2434	0.2430	0.2422	0.2410	0.2385	0.2324	0.2242	0.2140	0.2023
	(2015)										
	Akavci and										
	Tanrikulu	$\neq 0$	0.2367	0.2364	0.2359	0.2353	0.2338	0.2300	0.2249	0.2182	0.2102
	(2015)										
	\overline{Y} ounsi et al.										
	(2018)	$= 0$	0.2416	0.2412	0.2404	0.2392	0.2366	0.2305	0.2222	0.2121	0.2003
	Younsi et al.										
	(2018)	$\neq 0$	0.2371	0.2369	0.2364	0.2357	0.2342	0.2304	0.2252	0.2186	0.2105
	Present study	$= 0$	0.2461	0.2457	0.2449	0.2437	0.2412	0.2351	0.2269	0.2168	0.2051
	Present study	$\neq 0$	0.2357	0.2354	0.2350	0.2343	0.2328	0.2291	0.2240	0.2174	0.2094
$\overline{2}$	Mantari and										
	Soares (2016)	$\neq 0$	0.3927	0.3921	0.3908	0.3889	0.3849	0.3752	0.3621	0.3460	0.3273
	Mantari and										
	Soares (2016)	$= 0$	0.3810	0.3803	0.3790	0.3770	0.3730	0.3630	0.3497	0.3344	0.3165
	Akavci and										
	Tanrikulu	$= 0$	0.3896	0.3889	0.3877	0.3857	0.3817	0.3719	0.3588	0.3425	0.3237
	(2015)										
	Akavci and										
	Tanrikulu	$\neq 0$	0.3790	0.3787	0.3779	0.3768	0.3744	0.3684	0.3602	0.3496	0.3368
	(2015)										
	Younsi et al.				0.3847						0.3206
	(2018)	$= 0$	0.3867	0.3860		0.3828	0.3787	0.3689	0.3557	0.3394	
	Younsi et al.		0.3797	0.3793	0.3786	0.3774	0.3750	0.3691	0.3608	0.3501	0.3373
	(2018)	$\neq 0$									
	Present study	$= 0$	0.3939	0.3933	0.3920	0.3901	0.3860	0.3763	0.3632	0.3470	0.3282
	Present study	$\neq 0$	0.3774	0.3770	0.3763	0.3752	0.3728	0.3669	0.3587	0.3482	0.3355
$\overline{3}$	Mantari and										
	Soares (2016)	$\neq 0$	0.4418	0.4411	0.4396	0.4375	0.4330	0.4221	0.4074	0.3893	0.3683
	Mantari and	$= 0$	0.4286	0.4279	0.4264	0.4242	0.4196	0.4084	0.3934	0.3761	0.3558
	Soares (2016)										
	Akavci and										
	Tanrikulu	$= 0$								0.4383 0.4376 0.4361 0.4340 0.4294 0.4185 0.4036 0.3854 0.3642	
	(2015)										
	Akavci and										
	Tanrikulu	$\neq 0$	0.4265	0.4261	0.4252	0.4239	0.4212	0.4146	0.4053	0.3934	0.3789
	(2015)										
	Younsi et al.	$= 0$	0.4350	0.4343	0.4328	0.4307	0.4261	0.4151	0.4002	0.3819	0.3607
	(2018)										
	Younsi et al.	$\neq 0$	0.4273	0.4268	0.4260	0.4247	0.4220	0.4153	0.4059	0.3940	0.3795
	(2018)										
	Present study	$= 0$	0.4432	0.4425	0.4410	0.4389	0.4343	0.4234	0.4086	0.3904	0.3693
	Present study	$\neq 0$	0.4246	0.4242	0.4234	0.4221	0.4194	0.4128	0.4036	0.3918	0.3775

Table 7 Non-dimensional central displacement $\overline{w}(0) = G(h)w / h q_0$ and in-plane normal stress $\bar{\sigma}_x(0) = \sigma_x(0)/q_0$ of E-FGM plates subjected to uniformly distributed load.

Table 8 Comparison of the dimensionless deflection w^* of isotropic square plate subjected to uniformly distributed load.

10	1.2615	1.21169	1.2614	.2684	1.2543	.25412	1.2542	2531
15	.1627	1 1 1 7 2 5	1.1627	. 1694	1.1572	15711	1.1572	1.1562
20	1.0782	1.03638	1.0782	.0847	1.0740	1.07389	1.0740	

Table 9 Comparisons of non-dimensional displacements and stresses of simply supported FG rectangular plate (Al/Al₂O₃) resting on elastic foundation under uniform loads $(b = 3a, a/h = 10)$.

